

The University of Calgary  
Department of Chemical & Petroleum Engineering

ENCH 501: Transport Processes Quiz #7

November 25, 2003

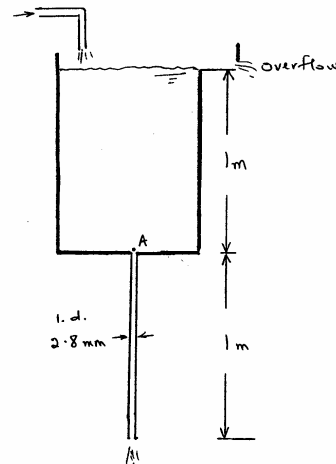
Time Allowed: 45 mins.

Name: \_\_\_\_\_

A liquid (density =  $930 \text{ kg/m}^3$ ) flows by gravity through a tube attached to the bottom of a large tank, as shown in the sketch below. The volume rate of the liquid out of the tube is measured as  $0.048 \text{ m}^3/\text{hour}$ . Neglecting entrance effects (i.e. assuming the flow is fully developed within the tube),

- (a) estimate the viscosity ( $\mu$ ) for the fluid.  
(b) Estimate the gauge pressure at point A (inlet of the tube).

**Hint:** Use Bernoulli equation with frictional losses allowed for.  
Also assume that the flow in the tube is laminar.  
Check this assumption.

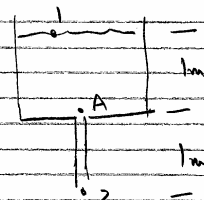


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Bernoulli equation

$$\frac{P_1}{\rho g} + z_1 + \frac{V_1^2}{2g} = \frac{P_2}{\rho g} + z_2 + \frac{V_2^2}{2g} + h_f$$

where  $h_f$  is head loss due to friction.



For the problem, the average velocity at exit, point 2 is

$$V_2 = \frac{Q}{A_2} = \frac{Q}{\pi R^2} = \frac{0.048 / 3600}{\pi (1.4)^2 (10^{-6})} = 2.1654 \text{ m/s}$$

$P_1 = P_2 = P_{atm}$ ; large tank  $\Rightarrow V_1 \approx 0$

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$z_1 - z_2 = 2 \text{ m}$  ( $\rho = \text{const.}$ )

Substitute into Bernoulli equation

$$2 = \frac{(2.1654)^2}{2(9.81)} + h_f \quad \text{or} \quad h_f = 1.761 \text{ m}$$

For laminar flow within the tube,

$$h_f = 64 \left( \frac{\mu}{D \rho g} \right) \left( \frac{L}{D} \right) \left( \frac{\bar{V}^2}{2g} \right)$$

$$1.761 = 64 \left( \frac{\mu}{D \rho g} \right) \left( \frac{1}{2.8(10^{-3})} \right) \left( \frac{(2.1654)^2}{2(9.81)} \right)$$

$$\therefore \frac{\mu}{D \rho g} = 3.22(10^{-4}) \quad \text{or} \quad Re = 3102 \quad (\text{turbulent flow})$$

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$$\mu = 2.8(10^{-3}) (2.1654) (932) (3.22)(10^{-4})$$

$$= 0.001818 \text{ Pa.s} \quad \text{or} \quad 1.82 \text{ mPa.s} \rightarrow$$

○ (b) For  $P_A$ , use Hagen-Poiseuille equation  
Equation 6.30 (Notes)

$$Q = - \pi \frac{\left[ \frac{dP}{dz} + \rho g \sin \beta \right] R^4}{8\mu}$$

$$= \pi \frac{[P_A - P_2 - \rho g L \sin \beta] R^4}{8L\mu} \quad \text{and } \beta = -\frac{\pi}{2}$$

$$Q = \pi \frac{[(P_A - P_2) + \rho g L] R^4}{8\mu L}; \quad P_A - P_2 \text{ is gauge pressure at A}$$

substitute numbers

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$$\frac{0.048}{3600} = \pi \frac{[P_{A_g} + 930(9.81)(1)] (1.4)^4 (10^{-12})}{8(1.818)(10^{-3})(1)}$$

$$P_{A_g} = 6,942.7 \text{ Pa}$$

$$= \frac{6.942}{101.325} \text{ atm} = 0.0685 \text{ atm}$$

Note: The hydrostatic head at A =  $h\rho g$   
 $= 1(930)(9.81) = 9,123.3 \text{ Pa.}$

$\therefore P_{A_g}$  is  $\angle$  hydrostatic head.

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